

Design and Fabrication of motorized automated Object liftingjack

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ABSTRACT: - With the increasing levels of technology, the efforts are being put to produce any kind of work thathas been continuously decreasing. The efforts required in achieving the desired output can be effectively and conomically be decreased by the implementation of better designs. Power screws are used to convert rotarymotion into reciprocating motion. An object lifting jack is an example of a power screw in which a small forceapplied ina horizontal plane isused to raise orlowera large load. Inthis fabricated model, an electric

motorwillbeintegratedwiththeobjectliftingjackandth eelectricityneededfortheoperationwillbetakenfromt he

d.cbatteryandtherebythemechanical advantage will be increased.

Keywords:-

Objectliftingjack, automation, limits witch, lead screw

I. INTRODUCTION

Our research survey in this regard revealed that in several automobile garages, revealed the facts thatmostly some difficult methods were adopted in lifting the vehicles for reconditioning, repair and maintenance. This fabricated model has mainly concentrated on this difficulty, and hence a suitable device has been designed, such that the vehicle and heavy objects can be lifted from floor land without the application of impact force. Thefabrication part of it has been considered with almost case for its simplicity and economy, such that this can beaccommodated as one of its essential tools on automobile garages. The object lifting jack has been developed tocater to the needs of small and medium automobile garages, which are normally man powered with minimumskilled labour. In most of the garages the vehicles are lifted by using screw jack. This needs high man power andskilled labour. In order to avoid all such disadvantages, the automated motorized object lifting jack has beendesigned in such a way that it can be used to lift the vehicle smoothly without impact very any force.

Theoperation is made simple so that even unskilled labour can use it with ease. The d.c motor is coupled with thelead screw by gear arrangement, the lead screw rotation depends upon the rotation of d.c motor. This is an era ofautomation where it is broadly defined as replacement of manual effort by mechanical power in all degrees ofautomation. The operation remains to be an essential part of the system although with changing demands onphysicalinput,

thedegreeofmechanizationisincreased.

Needfor automation

- Toachievemass production.
- Toreducehumaneffort.
- Toincreasetheefficiencyofthejack.
- Toreducetheworkload.
- Toreducetheproduction cost.
- Toreducetheproductiontime.
- Toreducethematerialhandling.
- Toreducethefatigueofworkers.

Principleofoperation

The lead screw is considered as an inclined plane with inclination α . When the load is being raised orlowered, following forces act at a point on this inclined plane.

Load(W):

Italwaysactsinverticallydownwarddirection. Normalreaction

(N):Itactsperpendicular(normal)totheinclinedplane. Frictional force (μ N): It acts opposite to the motion. When the load is moving the inclined plane, frictionalforce acts along the inclined plane in downward direction and when the load is moving down the inclined

plane, frictional force acts along the inclined plane in up ward direction.

Effort(P): It acts in a direction perpendicular to the load (W). It may act towards right to overcome thefrictionandraisetheload.

Whenloadisraised,



For an equilibrium of horizontal forces, $P = \mu N \cos \alpha + N \sin \alpha \dots (1)$ For an equilibrium of vertical forces, $W = N \cos \alpha - \mu N \sin \alpha \dots (2)$ Dividingequation(1)by(2)weget, $P = W (\mu \cos \alpha + \sin \alpha) \dots (3)\cos \alpha - \mu \sin \alpha$ The coefficient of friction μ is expressed as $\mu = \tan \theta \dots (4)$ Substitutingequation(4)in(3)weget, $P = W \tan(\theta + \alpha) \dots (5)$ ThetorqueTrequiredtoraisetheloadisgivenby,T=tan(

Thetorque Frequired toraise the load is given by, $T = tan(\theta + \alpha)...(6)$

Whenload islowered,

For an equilibrium of horizontal forces, $P = \mu N \cos \alpha - N \sin \alpha \dots (7)$ For an equilibrium of vertical forces, $W = N \cos \alpha + \mu N \sin \alpha \dots (8)$ Dividingequation(7)by(8)weget, $P = W (\mu \cos \alpha - \sin \alpha) \dots (9)\cos \alpha + \mu \sin \alpha$

Substituting equation (4) in (9) we get, $P = Wtan(\theta - \alpha)....(10)$

ThetorqueTrequiredtoraisetheloadisgivenby,T=tan(θ - α)(11)

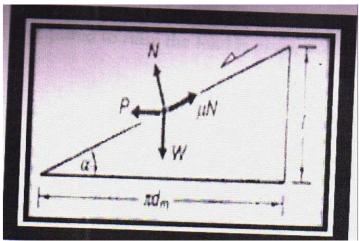


Fig 1:Forces actinginan inclinedplane

Componentsoffabricatedmodel

Themain partsoftheautomatedmotorized objectliftingjack areasfollows:

D.Cmotor

An electric motor is a machine which converts electrical energy into mechanical energy. Its action isbased on the principle that when a current carrying conductor is placed on a magnetic field, it experiences amagnetic force whose direction is given by Fleming's left hand rule. When a motor is in operation, it developstorque. This torque can produce mechanical rotation. D.C motors are also like generators classified into shuntwoundorserieswoundorcompoundwoundmoto rs.

LeadScrew

A lead screw is a portable device consisting of s screw mechanism used to raise or lower the load. Thelead screw can be short, tall, fat or thin depending on the amount of pressure they will be under and space thatthey need to fit into. It is made of various types of metals but the screw itself is made of lead.A large amount ofheat is generated init and long lifts cancause serious overheating. To retain the efficiency, it must be usedunder ambient temperatures, otherwise lubricants must be applied. These are oil lubricants intended to enhancethe equipment's capabilities. Apart from proper maintenance, to optimize the capability and usefulness of leadscrewitisimperativetoemployitaccordingtoitsde signandconstruction.

Batteries

Inisolated systems away from the grid, batteries are used forstorage of excess solarenergy whichcanbe converted into electrical energy. In fact for small units with output less than one kilowatt, batteries seem to beless the only technically and economically available storage means. Since both the photovoltaic system andbatteries are high in capital costs, it is necessary that the overall system be optimized with respect to availableenergyandlocaldemandpattern.

Ballbearing

This is a type of rolling element bearing that uses balls to maintain the separation between the bearingraces. The purpose of a ball bearing is to reduce rotational friction and support and radial and axial loads. Itachievesthisbyusingatleasttworacestocontaintheba llsandtransmittheloadsthroughtheballs.



Spur gears

These are designed to transmit motion and power between parallel shafts which are the most economical gears in the power transmission industry. T wotypes areusedinthismodel:

Internalspur gear

These spur gears are turned inside out. In other words, the teeth are cut into the inside diameter while theoutside diameter is kept smooth. This design allows for the driving pinion to rotate internal to the gear, which, inturn, allows for clean operation. Intended for light duty applications, these are gears always available only inbrass. When choosing a mating spur gear always remember that the difference in the number of teeth betweentheinternalgearandpinionshouldnotbeless than12or15.

1.3.5.2Externalspurgear

Perhaps the most often used and simplest gear system, external spur gears are cylindrical gears withstraight teeth parallel to the axis. They are used to transmit rotary motion between parallel shafts and the shaftsthat rotate in opposite directions. They tend to be noisy at high speeds as the two gear surfaces come into contactatonce.

LimitSwitch

It is a switch operated by the motion of a machine part or presence of an object. It is used for control of amachine, as safety interlocks, or to count objects passing a point. It is a

Specificationofparts D.Cmotor

Torque	10Kgcm
Speed	150rpm
Voltage supply	12V
Туре	D.C

1.4.3LargerGear 1.4.2LeadScrew

Outerdiameter(d _o)	13.7mm
Innerdiameter (d _i)	11mm
Mean diameter(d)	12.7mm
Pitch	2mm

SmallerGear

Addendumcirclediameter	73.45mm
Dedendumcirclediameter	61.80mm
Largerwidthoftooth	5.46mm
Smallerwidthoftooth	1.44mm
Depth ofcut	6.45mm
Thickness	12.95mm

Addendumcirclediameter	44.95mm
Dedendumcirclediameter	32.52mm
Largerwidthoftooth	4.76mm
Smallerwidthoftooth	1.14mm
Thickness	13.06mm

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electromechanical device that consists of an actuator mechanically linked to a set of contacts. When an object comes into contact with the actuator, thedevice operates the contacts to make or break an electrical connection. It is used in a variety of applications and environments because of their ruggedness, ease of installation, and reliability of operation. It can determine thepresence or absence, passing, positioning and end of travel of an object. It was first used to define the limit oftravelofanobject, hencethename'limitswitch.'

Controlswitch

It is used in order to start or stop the entire operation of the object lifting jack. The type of switch that is used is known as a toggle switch. The toggle switchis a class of electrical switches that are manually actuated by a mechanical lever, handle, or rocking mechanism. This is designed to provide the simultaneous actuation of multiplesets ofelectrical contacts, or the control of large amounts ofelectriccurrentormainsvoltages.

Controlcables

These are used in order to connect the battery to the motor andtheswitch.

Baseand Frame

A base for the entire set-up has also been made. The motor is mounted on an inverted U shaped supportframe.Ballrollersareattachedtofourends ofthebaseformovementandareelectricallycontrolled byswitch.



Roller

Torque	5 Kg cm
Speed	150rpm
Voltage supply	12V
Туре	D.C
Number	2

1.4.7Limit Switch 1.4.6 Base

Length	72.5mm
Breadth	42.5mm
Thickness	2cm
Materialused	Plywood

Number	2
Туре	Rollertype
Twowayvoltage supply	12V

ControlSwitch

Туре

DPCO(DoublePoleControl Off)

LoadPanel

Diameter	9.8cm
Thickness	5mm



Fig2:FabricatedmodelofObjectliftingjack

II. DESIGNCONSIDERATIONS

The load on the screw is the load which is to be lifted W, twisting moment M, between the screwthreads and force F at the handle to rotate the screw. The load W is compressive in nature and induces the compressive stress in the screw. It may also lead the screw to buckle. The load F produces bending and it ismaximum, when the screw is at its



maximum lift. The screw also experiences twisting moment due to F, theshearstressisalsoinducedinthescrewduetothetwist ingmomentbetweenthethreadsofscrewandnut. StepIProblemSpecification

It is required to design an object lifting jack for supporting the machine parts during their repair andmaintenance. It should be a general purpose jack with a load carrying capacity of 50 KN and a maximum liftingheightof0.3m.Thejackissooperatedbymeansof aD.Cmotor.

StepIISelectionofMaterials

- (i) The frame of the object lifting jack has complex shape. It is subjected to compressive stress. Grey cast iron isselected as the material for the frame. Cast iron is cheap and it can be given any complex shape withoutinvolvingcostlymachiningoperations.C astironhashighercompressivestrengthcompared withsteel.Therefore,itistechnicallyandeconomi callyadvantageous tousecastironfortheframe.
- (ii) Thescrew issubjected to torsional moment, compressive force and bending

moment.Fromstrengthconsideration,EN8issele ctedasmaterialforscrew.

(iii) There is a relative motion between the screw and the nut, which results in friction. The friction causes wearat the contacting surfaces. When the same material is used for these two components, the surfaces of bothcomponents get worn out, requiring replacement. This is undesirable. The size and shape of the screw make itcostlycomparedwiththenut. Thematerialusedforthenutisstainlesssteel.

StepIIIDesignofobjectliftingjack

Theobjectliftingjackisanintermittentlyused eviceandwearofthreadsisnotanimportantconsiderati on. Therefore, instead of trapezoidal threads, the screw is provided with square threads. Squarethreads have higher efficiency and provision can be made for self-locking arrangement. When the condition ofself-locking is fulfilled, the load itself will not turn the screw and descend down, unless an effort in the reversedirectionisapplied.

III. CALCULATIONS

Observed data:

Nominal diameter of screw, d = 13.7 mmCore diameter of screw, $d_c = 11$ mmPitchofscrewthread, p=2mm LoadW=20kg Coefficientoffriction, μ =0.15

Mean diameter of screw, $d_m = 12.7$ mmHelixangleofscrew, $\alpha = 2.68^{\circ}$

Tangential force required at the circumference of the screw to raise the load μ = tan ϕ = 0.15 p = W × tan α + tan ϕ = 40.2 N 1-tanα.tanø Torquerequired to operate the screw = $p \times d + \mu r_m$ 2 $=40.2\times(12.7/2)+(0.15\times200\times18)$ =825.27Nmm=0.8257Nm=8.5Kgcm Efficiencyof the screw= T_0/T_1 200×(12.7/2) $= 27\%0.15 \times 200 \times 18 + 200 \times (12.7/2)$ Forloweringload(P)=Wtan(α + ϕ) =W×tanα+tanø=19.826N 1-tan α.tanø Torque= $p \times d + \mu r_m W = 0.662 N$ 2 $\pi(d_c)^3$ T_1 , $\tau = 16T_1/$ 825.27 N/mm^2 3.15 Shear stress due to torque N/mm^2 Compressivestressduetoaxialload(σ_c)=W/A=2.10N/mm $\sqrt{\sigma_{c}^{\prime}+4\tau}$

Shear stress due to torque (σ_{cmax}) = 0.5 [σ_c +Maximum shear stress = 3.32 N/mm² < 40 N/mm²So,designissafe.

 $Spur Gear, \\ Gear Ratio = 1.75T_p = 16 \quad T_g = 28 \\ Velocityratio = 0.571N_p = 150 \qquad N_g = 85.714$



Ag=73.45 $A_{p}=44.95$ D_g=61.80 D_p=32.96 Y= 0.175-0.841/no.ofteeth $= \underline{\pi \times m \times T_p \times N_p} = 40\pi m 160$ $TakingC_s = 1$ $W_t = \underline{P \times C_s} = \underline{15.40 \times 1} = 122.55V$ 0.12566 =4.5 N/mm²<50 N/mm² $C_v = 3 = 3$ = 0.9593+v 3+0.12566 Yp=0.175-0.84=0.1224 16 Y_g=0.1449 $W_t = \sigma_{wp} \times b \times \pi \times m \times Y_p$ $15.40 = 60 \times 6 \times \pi \times 0.097 \times 3$ 0.1256 3+0.12566 46.2+1.93424=96.4521m³ m = 2.5 standardAddendum = 3.45 mmDedendum=8.2mm Centre distance between the shaft = 55 mmWeartoothload(W_w)= $D_p \times b \times Q \times K$ =40×12.5× (14/11)× 1.57 =1000N $DynamicLoad(W_d) = W_t + W_t$ $= \underline{21v(b_c + W_t)}$ = 43.3 N21 v +V = 0.341 m/sb = 12.5 mm $\sqrt{b_c + W_t}$ = 1063.41 + 1 $C = \underline{K}_{\underline{e}}$ Ep $E_{g}W_{d} = 825.4N$ SinceW_d<W_w.Designistrue. $W_s = \sigma_c \times b \times P_c \times y\sigma_c = 80 \text{N/mm}^2$ $= 12.5 \text{ mmP} = \pi \text{m}$ b Y=0.1224 $W_s = 958.185W_s > W_D$ Itis true.

IV. WORKINGMECHANISM

STEP 1 The lead acid battery is used to drive the d.c motor. The d.c motor shaft is connected to the spur gear. If power is driven to the d.c motor, it will run so that the spur gear also runs to slow down the speed of the d.cmotor. The object moving jack moves the lead screw upwards, so that the vehicle lifts from the ground. Thevehicle is lifted by using the lifting platform at the top of the jack. The motor draws power supply from thebattery. Thelifting and uplifting is done by changingt he battery supply to the motor.

STEP 2After pressing the DPCO switch, the circuit is completed and from battery power is transferred to themotor that is connected to the roller. Now the roller starts moving. Now controlling the two number of DPCOswitch which is connected to the two motors at the base the whole set-up is adjusted below the body which is being lifted.

STEP 3 Now pressing the DPCO switch to the circuit which is connected to the motor that is coupled to

theleadscrew,thecircuitiscompletedandvoltagefromt hebatteryispasstothemotor.Whentappingtheswitchto the positive pole, positive voltage is supplied to the d.c motor moves in clockwise direction and lead screwmoves indownwarddirection.

STEP 4When tapping the switch to the negative pole, negative voltage is supplied to the d.c motor moves

inanticlockwisedirectionandleadscrewmovesinupw arddirection.

STEP 5 Now when the lead screw moves to the maximum limit, the limit switch at the upper end getsactivated and the circuit gets cut-off. When the lead screw moves to the minimum limit, the limit



switch at thebottomendgetsactivatedandthecircuitgetscut-off.

V. CONCLUSION

Object lifting jacks are the ideal product to push, pull, lift, lower and position loads of anything from acoupleof kilograms to hundreds of tonnes. The need has long existed for animproved portable jack forautomotive vehicles. it is highly desirable that a jack become available that can be operated alternatively frominside the vehicle or from a location of safety off the road on which the vehicle is located. Such a jack should belight enough and be compact enough so that it can be stored in an automobile trunk, can be lifted up and carriedby most adults to its position of use, and yet be capable of lifting a wheel of a 4000-5000 pound vehicle off theground. Further, it should be stable and easily controllable by a switch so that jacking can be done from aposition of safety. It should be easily movable either to a position underneath the axle of the vehicle or someother reinforced support surface designed to be engaged by a jack. Thus, the product has been developed considering all the above requirements. This particular design of motorized automated object lifting jack willprovetobebeneficialinliftingandloweringofheav vloads.

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